1 498 007

(21) Application Nos. 3003/74 (22) Filed 22 Jan. 1974 6445/74 13 Feb. 1974

(23) Complete Specification filed 16 Jan. 1975

(44) Complete Specification published 18 Jan. 1978

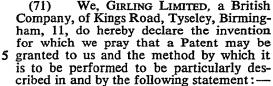
(51) INT. CL.2 F16D 55/22 65/02

(52) Index at acceptance

F2E 2N1Â1 2N1A4A2 2N1A5 2N1C3 2N1D16 2N1D2B 2N1D6A 2N1D6B 2N1D6C1 2N1D6C2 D24

(72) Inventor HEINRICH BERNHARD RATH

(54) IMPROVEMENTS RELATING TO DISC BRAKES



This invention relates to improvements in disc brakes of the kind in which a friction 10 pad assembly is adapted to be applied to one side of a rotatable disc by a support which may comprise a piston in the case of a directly actuated pad assembly and a part of a caliper or clamping member in 15 the case of an indirectly actuated pad assembly.

Disc brakes are liable to squeal under certain conditions, and various attempts have been made to prevent this by modification 20 of the mounting and application of the friction pad assemblies. For example, in brakes of the floating caliper type it has been proposed to clamp the outboard friction pad assembly to the caliper, and in 25 brakes of the fixed caliper opposed-cylinder type it has been proposed to offset the engagement between a piston and a friction pad assembly.

Furthermore, in our British Patent No. 30 925294 we have described and claimed a disc brake in which, for the purpose of eliminating or reducing squeal, there is interposed between the friction pad assembly and an hydraulically actuated piston 35 a shim of which a part is cut out over a portion of what would normally be the area of contact between the piston and the friction pad assembly, whereby when the piston is pressurised it applies a thrust to 40 the backing plate only over the remainder of that area.

According to our invention in a disc brake of the kind set forth a strip of material coated on both sides with a layer 45 of adhesive is interposed between the friction pad assembly and the support to locate the friction pad assembly without actually clamping it and restrict the damp relative movement, radial or circumferential, between the pad assembly and the 50 support

(11)

By attaching the pad assembly to the support, the vibrating mass is increased. This modifies the vibration frequency of the pad assembly by reducing it to a 55 lower and less audible level.

The strip may comprise a single strip of metal. Alternatively the strip may be of laminated construction provided with at least two separate metal layers interconnected by resilient material. Such a construction damps out and prevents amplification of high frequency vibrations.

The adhesive employed is such that the bond formed by the adhesive between the 65 friction pad assembly and the support is sufficient to prevent relative movement of the pad assembly in use but is not so great as to prevent withdrawal of the pad assembly when the pad assembly has to be 70 replaced.

Suitable adhesives are available on the market, a particularly suitable adhesive being based on uncured rubber embedded in silk.

In the initial assembly of the brake at a factory the coated strip may be supplied in the form of a roll from which appropriate lengths are cut off automatically during assembly.

A suitable material for the strip is aluminium, but various other metals or alloys may be employed.

When the strip comprises a shim having a cut-out and the support comprises a 85 piston working in an hydraulic cylinder the cut-out is so constructed and arranged that when the pad assembly is in its position of use in the brake the centre of area of the engagement between the piston 90



CID: <G

1498007A L >

25

40

and the pad assembly lies on the side of the axis of the piston opposite to that, with which any given point on the brake disc first comes into alignment in the normal 5 forward direction of disc rotation.

The shim may be so shaped that it can only be inserted the right way round, or it may carry a marking, such as an arrow, to ensure that it is fitted correctly.

Some embodiments of our invention are illustrated in the accompanying drawings in which:

Figure 1 is a transverse section through

a metal locating strip;

Figure 2 is a section similar to Figure 1 of a strip of laminated construction; Figure $\bar{3}$ is a section through an hydraulic piston with a locating strip attached to it;

Figure 4 is a yoke of a disc brake pro-

20 vided with a locating strip; Figure 5 is a section through a part of

an hydraulically-operated disc brake with a locating strip interposed between an hydraulic piston and a friction pad

assembly; Figure 6 is a side elevation of a friction pad assembly with a locating strip attached thereto;

Figure 7 is a view similar to Figure 6 of a locating strip of different configura-

> Figure 8 is a view similar to Figure 6 including a pair of locating strips;

Figure 9 is another view similar to Figure 35 6 but including yet another configuration of locating strip;

Figure 10 is a plan of a disc brake; Figure 11 is a view on an enlarged scale

of a part of the disc brake; and Figure 12 is a side elevation of shim for use in the disc brake of Figures 10 and

The locating strip 1 illustrated in Figure 45 1 comprises a strip 2 of a substantially incompressible material, suitably a metal, for example aluminium, coated on both side faces with a layer of adhesive 3. Conveniently the strip comprises a length cut

50 from a roll as required with the coated faces protected by "peel-off" layers of oiled paper 4. The adhesive may be of any commercially available type but an adhesive based on uncured rubber embedded in silk 55 is particularly suitable.

In the modified construction illustrated in

Figure 2 the metal strip 2 is of laminated construction comprising two metal layers 5 which are interconnected by resilient mate-

60 rial 6 which may comprise an additional layer of adhesive. The outer faces of the metal layers are coated as before with the layers of adhesive 3 which, in turn, are protected by the "peel-off" layers.

The locating strip 1 is adapted to be in-

stalled in a vehicle disc brake of the kind in which a friction pad assembly is adapted to be applied to one or each side of a rotatable disc by a support with the strip 1 interposed between the pad assembly and the 70

As illustrated in Figure 3 the support comprises an hydraulic piston 7 adapted to work in an hydraulic cylinder of the disc brake to apply a friction pad assembly 75 directly to an adjacent face of the disc. The strip 1 is applied directly to the piston during assembly of the brake after removal of one of the "peel-off" layers so that the strip is attached to the piston by the adhesive. 80 The other "peel-off" layer is removed before the pad assembly is inserted into the brake so that the piston 7 is secured to the

pad assembly by means of the locating strip. In the construction of Figure 4 the sup- 85 port comprises a yoke 8 having opposed interconnected portions 9 and 10 disposed on opposite sides of the disc. The yoke 8 is slidably guided for axial movement in a stationary member in a direction to 90 apply the brake in response to operation of actuating means. The actuating means act on the portion 9 in a direction to urge the portion 10 towards the disc. This indirectly urges a friction pad assembly 95 into engagement with the disc. As illustrated the locating strip 1 is normally applied to the portion 10 and is received within a recess 11 in which the indirectly actuated friction pad assembly is at least partially 100 received to locate it at least substantially against movement in a circumferential direction with respect to the yoke.

In the embodiment of Figure 5 the support again comprises the hydraulic piston 105 7 for applying a friction pad assembly 12 to a rotatable disc. The friction pad assembly 12 comprises a rigid backing plate 13 carrying on one face a pad 14 of friction material and the locating strip 1 is housed 110 within a recess 15 in the other face of the backing plate.

The locating strip 1 and the recess 15 may be of any convenient complementary outline. For example, as shown in Figure 115 6, the strip I and the recess 15 are circular. In Figure 7 the recess comprises a trans-

In another construction illustrated in Figure 8 strips 1 are applied diagonally to a 120 planar face of the backing plate 13 so that the piston 7, shown in chain dotted outline, acts on the pad assembly only over four angularly spaced contact areas 16.

In the construction shown in Figure 9 a 125 single strip 1 is applied to a planar face of the backing plate 13 for engagement by the support portion 10 of the yoke 8.

The disc brake illustrated in Figures 10 to 12 comprises a caliper 17 of generally U 130

outline straddling a portion of the peripheral edge of a rotatable disc not shown. Friction pad assemblies 18 for engagement with opposite faces of the disc are guided 5 in the caliper 17 for movement towards and away from the disc and are adapted to be applied to the disc by opposed pistons 19 working in hydraulic cylinders 20 in the caliper 17. Each friction pad assembly 18 10 comprises a pad of friction material 21 carried by a rigid backing plate 22, and there is interposed between each piston 19 and each backing plate a shim 23. A part 24 of each shim is cut out over a portion 15 of what would normally be the area of contact between each piston 19 and the backing plate 22 on which it acts so that when the pistons are pressurised they apply thrust to the backing plate 22 only over the remain-20 der of those areas. The shim 23 comprises a metal strip 2 which may be either a single member as illustrated in Figure 1 or of laminated construction as shown in Figure 2. In both cases the strip is coated with the 25 adhesive layers 3 to locate the pad assemblies 18 and restrict and damp any relative movement, radial or circumferential between the pistons 19 and the pad assemblies

As shown in Figure 12 the shim 23 may be shaped to fit into the brake, and it may carry marking such as an arrow 25 to ensure that it is fitted correctly.

WHAT WE CLAIM IS:

1. A disc brake of the kind set forth in which a strip of material coated on both sides with a layer of adhesive is interposed between the friction pad assembly and the support to locate the friction pad assembly 40 without actually clamping it and restrict and damp relative movement, radial or circumferential between the pad assembly and the support.

2. A disc brake as claimed in Claim 1, 45 in which the strip comprises a single strip

of metal.

3. A disc brake as claimed in Claim 1, in which the strip is of laminated construction comprising at least two separate metal 50 layers, and resilient material interconnect-

ing the metal layers.

4. A disc brake as claimed in any of Claims 1 to 3, in which the adhesive employed is such that the bond formed by the 55 adhesive between the friction pad assembly and the support is sufficient to prevent relative movement of the pad assembly in use but is not so great as to prevent withdrawal of the pad assembly when the pad 60 assembly has to be replaced.

5. A disc brake as claimed in any preceding claim in which the adhesive includes

uncured rubber embedded in silk.

6. A disc brake as claimed in any pre-65 ceding claim in which the coated strip is adapted to be protected on opposite sides by protective layers which are removed before the strip is inserted into the brake.

7. A disc brake as claimed in Claim 6, in which the protective layers comprise layers 70 of oiled paper which are adapted to be 'peeled off"

8. A disc brake as claimed in any preceding claim, in which the coated strip is

cut from a roll.

9. A disc brake as claimed in any preceding claim in which the pad assembly comprises a friction pad carried by one face of a rigid backing plate and the strip is received within a recess in the other face 80

of the backing plate.

10. A disc brake as claimed in any of Claims 1-8 in which the strip comprises a shim having a cut-out so constructed and arranged that when the pad assembly is in 85 its position of use in the brake the centre of area of the engagement between the support, which comprises a piston, and the pad assembly lies on the side of the axis of the piston opposite to that with which any 90 given point on the brake disc first comes into alignment in the normal forward direction of disc rotation.

A disc brake as claimed in any of Claims 1-9, in which the support comprises 95 an hydraulic piston working in an hydraulic

cylinder.

12. A disc brake as claimed in any of Claims 1 to 8, in which the support comprises a portion of a yoke which acts on 100 the friction pad assembly to urge it into engagement with the disc in response to operation of actuating means acting on another portion of the yoke disposed on the opposite side of the disc.

13. A disc brake of the kind set forth incorporating a locating strip substantially as described with reference to and as illustrated in Figure 1 of the accompanying

drawings.

14. A disc brake of the kind set forth incorporating a locating strip substantially as described with reference to and as illustrated in Figure 2 of the accompanying drawings.

15. A disc brake of the kind set forth having an hydraulic piston provided with a locating strip substantially as described with reference to and as illustrated in Figure 3

of the accompanying drawings.

16. A disc brake of the kind set forth incorporating a yoke provided with a locating strip substantially as described with reference to and as illustrated in Figure 4 of the accompanying drawings.

17. An hydraulically-operated disc brake substantially as described with reference to and as illustrated in Figure 5 of the accom-

panying drawings.

18. A disc brake of the kind set forth 130

1498007A__I_>

105

120

125

incorporating a friction pad assembly substantially as described with reference to and as illustrated in Figure 6 of the accompanying drawings

panying drawings.

19. A disc brake of the kind set forth incorporating a friction pad assembly substantially as described with reference to and as illustrated in Figure 7 of the accompanying drawings.

20. A disc brake of the kind set forth incorporating a friction pad assembly substantially as described with reference to and as illustrated in Figure 8 of the accompanying drawings

ing drawings.

15 21. A disc brake of the kind set forth

incorporating a friction pad assembly substantially as described with reference to and as illustrated in Figure 9 of the accompanying drawings.

ing drawings.

22. A disc brake for a vehicle substan- 20 tially as described with reference to and as illustrated in Figures 10 and 12 of the accompanying drawings.

BARKER, BRETTELL & DUNCAN,
Chartered Patent Agents,
Agents for the Applicants,
138 Hagley Road,
Edgbaston,
Birmingham, B16 9PW.

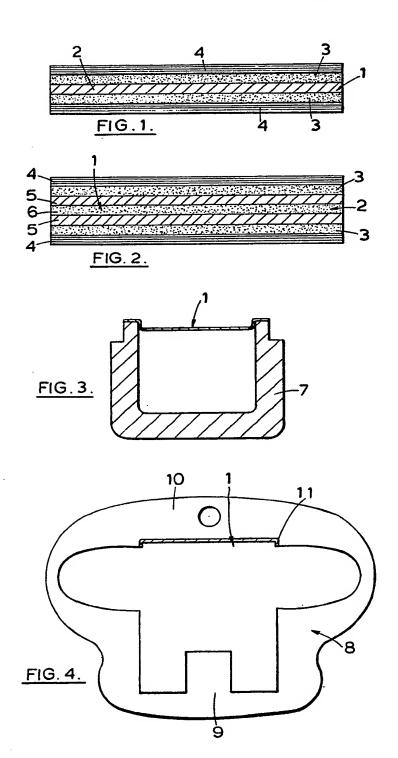
Printed for Her Majesty's Stationery Office by The Tweeddale Press Ltd., Berwick-upon-Tweed, 1978. Published at the Patent Office, 25 Southampton Buildings, London, WC2A 1AY, from which copies may be obtained.

OCID:_<GB____149

1498007A I

This drawing is a reproduction of the Original on a reduced scale.

SHEET |



BEST AVAILABLE COPY

This drawing is a reproduction of the Original on a reduced scale.

SHEET 2

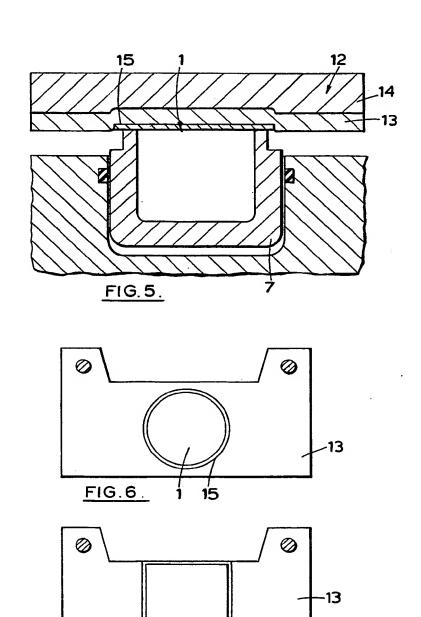
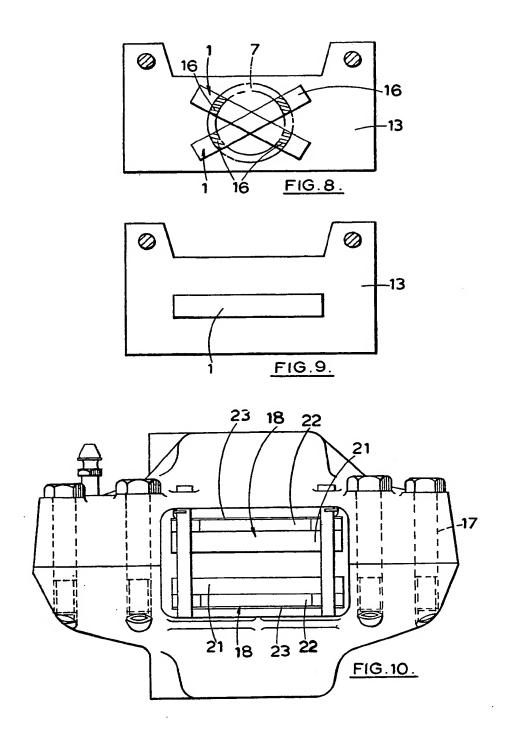


FIG. 7.

This drawing is a reproduction of the Original on a reduced scale.

SHEET 3



BEST AVAILABLE COPY

1 498 007

COMPLETE SPECIFICATION

4 SHEETS This drawing is a reproduction of the Original on a reduced scale.

SHEET 4

